

Effect of EGR on Performance and Emissions of C.I Engine Operated on Alternate Fuels: A Review

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Abstract—Increasing environmental concern and stringent emission norms are alarming the situation to look for appropriate emission reduction technologies. Out of various technologies, exhaust gas recirculation (EGR) is found to be a prominent tool to reduce and control the oxides of nitrogen (NO_x) emissions. This paper focuses on the effect of EGR technique on C.I engines fueled with different alternative fuels i.e. Biodiesel (karanja, jatropha), CNG. EGR lowers oxygen availability in the combustion chamber which reduces flame temperature. Application of EGR in diesel engine causes higher soot emissions. It leads to other engine problems like degradation of lubricating oil, higher carbon deposits and enhanced engine wear which are not favorable in long term engine running. It is found that application of EGR significantly reduces NO_x emissions with slight increase in other emissions like CO, HC and PM. However, with the variation in EGR% they can be controlled up to an extent. No significant variation in performance was reported by most of the authors. In order to counter the adverse effects of EGR, people have suggested to use augmented technologies along with EGR like DOC and DPF. Moreover, biodiesel blends are also found to be in good relationship with EGR from emissions point of view. Use of biodiesel with EGR results in lower CO and HC exhaust emissions. The loss in BTE could be overcome by increase in injection pressure and optimized injection timing for biodiesel fuel. Overall it is concluded that use of EGR with biodiesel could be a prominent solution for increasing environmental pollution arose due to diesel engines.

Keywords: Exhaust gas recirculation, Hot EGR, NO_x emission, smoke, biodiesel blend, injection pressure

1. INTRODUCTION:

Diesel engines are widely used because of higher thermal efficiency, lower fuel consumption, lower carbon monoxide (CO) and hydrocarbon (HC) emissions, etc. However, environmental issues have become increasingly prominent with the increment of diesel engines ownership. NO_x emitted from diesel engines is increasingly injurious to human health and the environment such as photochemical smog and acid

rain. As a result, many countries have enacted more stringent regulations to reduce it [1].

Although, major constituents of diesel exhaust include carbon dioxide (CO_2), water vapor (H_2O), nitrogen (N_2), and oxygen (O_2); carbon monoxide (CO), hydrocarbons (HC), oxides of nitrogen (NO_x), and particulate matter (PM) are present in smaller but environmentally significant quantities. In modern diesel engines, first four species normally consist of more than 99% exhaust, while last four (the harmful pollutants) account for less than 1% exhaust. NO_x comprises of nitric oxide (NO) and nitrogen dioxide (NO_2) and both are deleterious to humans as well as environmental health. NO_2 is more toxic than NO. It affects human health directly and is precursor to ozone formation, which is mainly responsible for smog formation [2]. The principal source of NO is the oxidation of atmospheric nitrogen. NO_x is formed during the combustion due to high temperature [3]. The different methods are available for the reduction of NO_x . The cetane improvers can reduce NO_x but the addition of cetane improvers increase the cost of fuel and the possibilities of auto oxidation. Retarded injection is the method used for reduction in NO_x for Compression ignition engines, but the method leads to increase in Brake specific fuel consumption and emissions. Water injection leads to corrosion and increase in the weight due to water storage [4]. The exhaust gas recirculation (EGR) technique is gaining widespread use as one of the most efficient methods or technique for reduction of nitrogen dioxide (NO_x). This involves the circulation of part of the exhaust gas along with the intake fresh air charge, into the combustion chamber of the diesel engine for combustion together with the fuel injected in the normal power cycle of the diesel engine. EGR is an emission control system which allows significant reduction in NO_x gases in almost all types of diesel engines including Light Duty, Medium Duty and Heavy-Duty engine use

or applications, and primary engines as low as two strokes used in marine operations and applications [5].

To study the effects of EGR on the performance and emissions of automotive engines, the system shown in Fig. 1 gives an example of the EGR system.

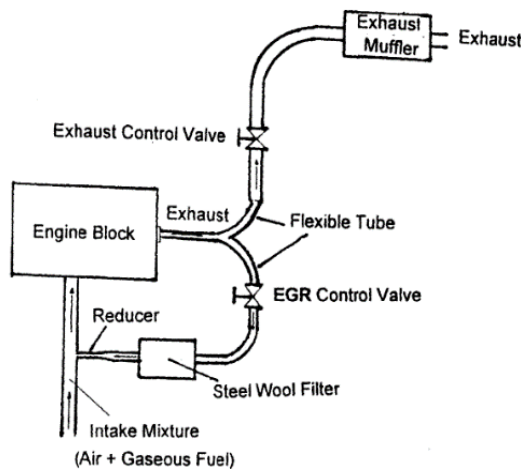


Figure 1 EGR system [6]

There are two ways to use exhaust in engine cylinder i.e. externally and internally. Use of variable valve timing (VVT) or other mechanisms to retain a certain fraction of exhaust from a preceding cycle is known as Internal EGR, whereas in external EGR the arrangement of external pipe could help to use exhaust by means of the pressure differential between the exhaust gas and inlet air.

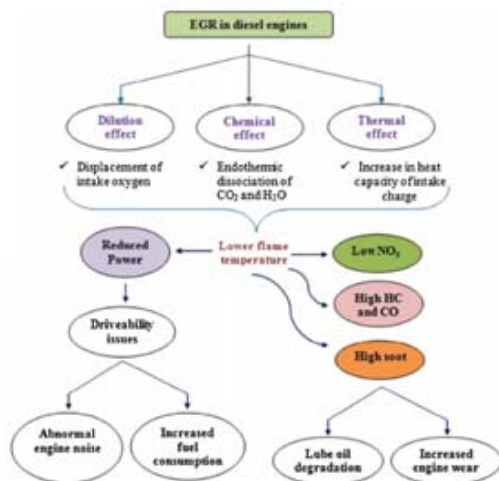


Figure 2: Effects of EGR on diesel combustion and pollutant formation [7]

To suppress NOx formation some of the fresh air is displaced by EGR which acts as a heat sink. The higher heat capacity

of these diluents results in lower combustion rate and temperature rise, which reduces the peak cylinder gas temperature. A pictorial representation of the EGR effect on diesel combustion is provided in fig. 2 [7]. When a part of this exhaust gas is re-circulated to the cylinder, it acts as a diluent to the combustion mixture. This also reduces the oxygen concentration in the combustion chamber. The specific heat of exhaust gas is much higher than that of fresh air. Hence EGR increases the specific heat of the intake charge, thus decreasing the temperature rise for the same heat release in the combustion chamber [8]. With EGR, the mean specific heat capacity of the engine inlet charge will be higher compared to that of air. This rise in specific heat capacity value is due to both carbon dioxide and water vapor having higher heat capacity values than nitrogen and oxygen. This rise in the mean specific heat capacity of the inlet charge will be called the thermal effect of EGR [9].

When EGR is applied, the engine intake consists of fresh air and recycled exhaust. The percentage of recycled gases is commonly represented by an EGR ratio, i.e. the mass ratio of recycled gases to the whole engine intake. The fresh air intake contains negligible amounts of CO2 while the recycled portion carries a substantial amount of CO2 that increases with EGR flow rate and engine loads. Notably, CO2 is merely a combustion product. Thus, it is intuitive and practical, to measure EGR ratio by comparing the CO2 concentrations between the exhaust and intake of the engine [10].

$$\text{EGR rate} = 100 \times (m_{\text{EGR}}) / (m_{\text{air}} + m_{\text{EGR}}) [11]$$

$$\text{EGR rate} = 100 \times (V_{\text{EGR}}) / (V_{\text{air}} + V_{\text{f}} + V_{\text{EGR}}) [12]$$

$$\text{EGR rate} = 100 \times (Q_{\text{without EGR}} - Q_{\text{with EGR}}) / Q_{\text{without EGR}}$$

$$\text{EGR rate} = 100 \times ([\text{CO}_2]_{\text{intake}} - [\text{CO}_2]_{\text{ambient}}) / ([\text{CO}_2]_{\text{exhaust}} - [\text{CO}_2]_{\text{ambient}}) [13]$$

2. EFFECT OF EGR ON ENGINE PERFORMANCE

There are so many studies that have been done on hot EGR and it has been reported that the EGR affected the performance and combustion characteristics. The variation of BTE with load for diesel operated engine at constant speed is lower at all loading conditions when operated with EGR and compared to without EGR [14]. As increase in EGR percentage the BTE decreases marginally. The possible reason may be the oxygen deficiency at higher load, which leads to incomplete combustion [15]. Another experiment done on Diesel - piloted biogas engine and found that as the engine load increases the brake thermal efficiency increases. EGR up to about 20% the BTE has increased and then started to fall. This is because of re-burning of HC that enters the combustion chamber with the recirculation of exhaust gases and EGR increases intake charge temperature which increases the rate of combustion. It is observed that BTE decreases if too much exhaust gas is recirculated; this is due to the higher exhaust replacing too much oxygen [16].

An experiment conducted with *Jatropha curcas* biodiesel on C.I. engine with varying load, injection pressure, biodiesel blend percentage and EGR rate and investigated the performance of brake thermal efficiency. The experimentation with 300 to 500 FIP for 75% and 100% load and the addition of blends B10, B20 has improved the BTE without EGR. Biodiesel contains oxygen which helps in better combustion which result into higher efficiency. A slight increase in BTE is observed when EGR rates of 15% is applied for blends B10 and B20 at 75% and 100% load with same condition of 300 bar FIP. 6.6% higher efficiency is obtained when operated at B20 at 15% EGR with 75% load and 300 bar FIP conditions as compared to diesel engine. This increase in efficiency is due to the re-burning of the hydrocarbon which mix with fresh air and goes into combustion chamber via intake manifold. The maximum improvement in performance is observed is 9.12% at full load (500 bar FIP) for B20 blend with 15% EGR in comparison with diesel fuel [17].

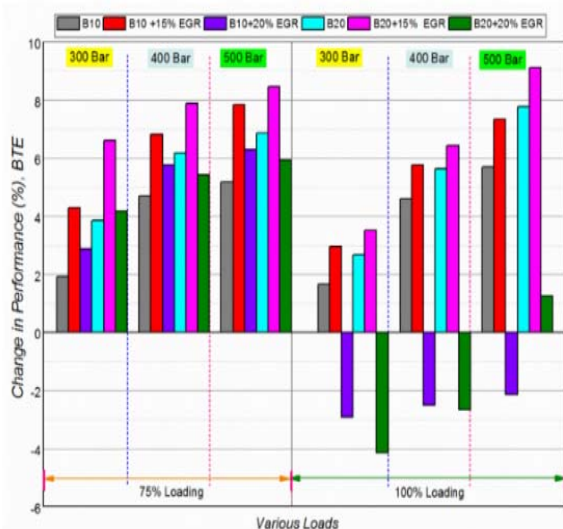


Figure 3 BTE 75% and 100% loads at 300,400,500 FIP [17]

The experiment performed using diesel and different blends of *Jatropha* biodiesel fuel with EHN (ethyl hexyl nitrate) as cetane improver with a variation in % mass fraction of EGR. The BTE is found to be increasing with the increase in biodiesel at a fixed EGR% and there is an increase in the rate of BTE at 50% load when compared to full load. Since at full load oxygen deficient operation under EGR is responsible for marginal increase in BTE. The increase in BTE with the increase in the percentage of EGR is due to the increased combustion velocity, as EGR increases the intake charge temperature [18].

The BSFC and BTE are both depended on each other if one increases than other decreases and vice versa. As the injection timing is advanced from 19 to 27 BTDC, for CNG-HOME fuel the BTE increases for 80% and 100% loads. This is due to the available time of burning for CNG fuel is more. The engine performance was smooth, and the ignition delay reduced

through the advanced injection timing but tended to incur a slight increase in fuel consumption. At 100% load, the BTE is lower compared with 80% engine operation. An improvement in BTE was achieved by advancing the injection timing. Maximum pressure and pressure rise rate are higher for the advanced injection timing [19].

Temperature of exhaust gas was observed to be lower in case EGR operated engine. The possible reason for this temperature reduction may be stated as relatively lower availability of oxygen molecule and higher specific heat of intake diluents [20]. The lower EGT with EGR rate signifies reduction of NO_x emission. As the EGR rate increases the volumetric efficiency decreases. The mass flow rate of intake air reduces because of EGR implementation the mixed intake air temperature increases and thus the density of air will decrease, and volumetric efficiency drops [21][22].

3. EFFECT OF EGR ON ENGINE EMISSIONS

DOC, DPF, SCR, and EGR are some of the pre-treatment/post-treatment systems designed to treat the exhaust gases of diesel engines, especially PM and NO_x. The three major causes of formation of NO_x are high combustion temperature, availability of excess oxygen, and duration of availability of excess oxygen. EGR, a widely used pre-treatment technique, reduces NO_x emissions from diesel engines by lowering oxygen concentration and flame temperature of the working fluid in the cylinder. As the EGR rate increases the NO_x emission decreases in diesel engine. At lower load percentage of NO_x reduction is higher than that of higher load in diesel engine [23]. The reduction of oxygen availability due to the displacement of some of the oxygen in the fresh intake air charge by the recirculated exhaust gas. This causes a reduction in the local flame temperature because of the spatial broadening of the flame due to the reduction in the oxygen molar fraction. Also, there is the thermal effect due to the increase in the average specific heat capacity of the gases in the combustion zone, since the recirculated exhaust gas contains CO₂ and H₂O with higher specific heat than that of air. Finally, there is a reduction in the combustion temperature due to endothermic chemical reactions, such as CO₂ and H₂O dissociation [24]. NO_x emissions for the conventional diesel engine were higher than the dual fuel engine operated with JOME and CNG at lower and intermediate loads. This was because of the higher in-cylinder temperature in the conventional diesel engine. At higher load, due to improvement in gaseous fuel utilization, higher in-cylinder temperature and extra availability of oxygen in JOME increases NO_x emissions in the dual fuel engine. Hence, at high load, the dual fuel engine emitted higher NO_x compared to the conventional diesel engine [25].

It is noticed that CO₂, CO and THC emissions increase with the increase of EGR rate. This trend is explained by the fact that the fresh intake air contains negligible amounts of CO₂, while the EGR fraction carries a substantial amount of CO₂, which is

increased with EGR flow rate and engine load. This increment becomes more relevant for higher EGR rates and engine loads[26]. Percentage of CO emission for diesel fuel has linearly increased overall 6.2% due to increasing engine speed when running with EGR. While in palm-biodiesel, it is found that CO emission has increased to a 9.2% corresponding with the increasing engine speeds under EGR mode. Percentages of CO emission from palm-biodiesel were higher than diesel when operating with EGR due to the some of the oxygen present in the inlet charge is replaced with recirculates exhaust gas that causes incomplete combustion. Furthermore, the increasing cetane number in palm-biodiesel helps to reduce the CO concentration in the exhaust piping. It was illustrated that palm-biodiesel has the highest rate emitting UHC with nearly 12% as compared to conventional diesel when operating with EGR due to the reduction of oxygen in the inlet charge, replaced by higher amount of carbon dioxide (CO_2) into the cylinder. It was demonstrated that CO_2 emission was increased when operating with the increasing engine speeds with palm-biodiesel with 9.4% under EGR mode[27].

CO emissions from sunflower methyl ester biodiesel blend with EGR and without EGR in C.I. engine is investigated and noticed that the CO increases with increase in load and EGR rate. However, CO emissions of SFME were comparatively lower. Higher values of CO were observed at full load for both diesel and biodiesel fuels with EGR. For biodiesel, the excess oxygen content is believed to have partially compensated for the oxygen deficient operation under EGR. Dissociation CO_2 to CO at peak loads where high combustion temperature and comparatively fuel rich operation exists, can also contribute to higher CO emissions[28].

The particulate matter (PM) is basically composed of soot and accounts for the smoke. The formation of opaque smoke ensues under the air deficit conditions which locally exist in the engine cylinder and increases as the air/fuel ratio declines. Smoke emission increases as the EGR% increases in diesel engine. Higher rate of increase in smoke emission was found at higher loads. This may be due to the reason that the charge becomes richer in fuel at higher load. The KOME and its blends showed slight reduction in smoke emission due to oxygen enrichment. The B50 and B100 KOME showed an average reduction of 7.17 and 11.8 % in smoke emission compared with neat diesel fuel[29]. Soot emission during combustion for various n-butanol–diesel blends at different EGR rates are investigated and noticed that as butanol content increases in the blends results in the reduction of soot due to a higher oxygen/carbon ratio[30].

4. EFFECT OF EGR ON ENGINE PARTS

Figure 4 shows the physical conditions of various vital engine parts like cylinder head, injector tip, and piston crown respectively of engine operated with EGR (a) and without EGR (b) on which carbon deposits take place due to the direct

expose to the combustion in-cylinder liner. Engine operated with EGR have the higher Carbon deposits on the various parts of the engine rather than the engine operated without EGR[31].



Figure 4 Carbon deposits on engine parts [31]

Engine parts give a qualitative estimation of amount of soot generation inside the engine by means of soot deposits on it and it was observed that the normally operated engine has lower soot emission as compared to engine operated with EGR system [32]. It has been observed that using EGR the extent of wear of top ring in the engine is lower than normal operating engine. This may be due to the lower temperature of the combustion chamber of the engine using EGR. However, using EGR the second and third compression ring and oil ring is comparatively having the higher wear rate. This may be due to the presence of higher amount of soot and wear debris in the lubricating oil as compared to the engine using without EGR [33].

5. CONCLUSIONS

- For reducing the NO_x emission EGR is a very useful technique for diesel fueled C.I. engine with slightly compromising with the efficiency and other emissions.
- The recirculation of exhaust gas to the combustion chamber displaces the oxygen content in intake air.
- Exhaust gases lower the oxygen concentration in combustion chamber and increase the specific heat of the intake air mixture, which results in lower flame temperatures.
- The emissions and performance of C.I. engine gets affected differently by lesser flame temperature and reduced oxygen content.
- In diesel engine at lower load the brake thermal efficiency can be marginally decrease or remain unaffected by increasing in EGR% up to 20%.

- Normal trend of brake thermal efficiency in diesel fuel operated engine is as increase in EGR% up to 30% the BTE will reduced slightly.
- Diesel fueled C.I. engine will cause slightly more CO and HC but higher smoke emissions with significant reduction in NO_x emission.
- 20% EGR can be used for reduction in NO_x emissions in diesel engine with approximately same BTE but slightly increase in HC & CO with higher smoke emissions
- For CO, HC & Smoke reduction biodiesel blends can be used with EGR which reduces all three emissions compare to diesel operated engine
- SFME blend B20 and 15% EGR rate culminates into NO_x reduction (25%) and HC & CO emissions decreased by 5% and 10%.
- But use of biodiesel blends will lead to decrease in BTE and increase in BSFC compare to diesel fueled engine
- For increasing the BTE the injection pressure is increased, and it is found that the HC, CO & Smoke emissions also reduced but penalty in NO_x emissions as compared with the standard injection pressure with EGR in C.I. engine
- Reduction in emission as well as improvement in performance were found in combination of biodiesel blend (B20) with EGR (15%) and FIP (500bar)

With the use of EGR system, amount of carbon deposition on various parts of engine has increased significantly

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